



(12) **United States Patent**
Ohta et al.

(10) **Patent No.:** **US 9,050,223 B2**
(45) **Date of Patent:** **Jun. 9, 2015**

(54) **LIFTING APPARATUS AND BED PROVIDED WITH THE SAME**

(71) Applicant: **Panasonic Corporation**, Osaka (JP)

(72) Inventors: **Akihiro Ohta**, Osaka (JP); **Shohei Tsukada**, Hyogo (JP); **Yohei Kume**, Osaka (JP); **Tomohiro Shimoda**, Tokyo (JP); **Hideo Kawakami**, Osaka (JP)

(73) Assignee: **Panasonic Intellectual Property Management Co., Ltd.**, Osaka (JP)

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

(21) Appl. No.: **14/081,248**

(22) Filed: **Nov. 15, 2013**

(65) **Prior Publication Data**

US 2014/0137328 A1 May 22, 2014

(30) **Foreign Application Priority Data**

Nov. 16, 2012 (JP) 2012-251750
Oct. 17, 2013 (JP) 2013-216144

(51) **Int. Cl.**

A61G 7/12 (2006.01)
B66F 7/06 (2006.01)
A61G 7/018 (2006.01)
A61G 7/012 (2006.01)
A61G 7/015 (2006.01)
A61G 7/10 (2006.01)
A61G 7/16 (2006.01)
A61G 7/053 (2006.01)
A61G 7/05 (2006.01)

(52) **U.S. Cl.**

CPC **A61G 7/018** (2013.01); **A61G 7/012** (2013.01); **B66F 7/0691** (2013.01); **A61G 7/015** (2013.01); **A61G 7/053** (2013.01); **A61G**

7/1019 (2013.01); **A61G 7/1046** (2013.01); **A61G 7/1057** (2013.01); **A61G 7/16** (2013.01); **A61G 2007/0518** (2013.01); **A61G 2007/0524** (2013.01); **A61G 2203/14** (2013.01)

(58) **Field of Classification Search**

CPC **A61G 7/018**; **A61G 7/012**; **A61G 7/015**; **A61G 7/053**; **A61G 7/1019**; **A61G 7/1046**; **A61G 7/1057**; **A61G 7/16**; **A61G 2007/0518**; **A61G 2007/0524**; **A61G 2203/14**; **B66F 7/0691**
USPC **5/611**, **11**, **86.1**, **610**, **616**, **109**, **613**, **5/618**, **617**, **600**; **254/160**
See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

5,303,437 A * 4/1994 Hung 5/613
5,365,622 A * 11/1994 Schirmer 5/611
5,404,603 A * 4/1995 Fukai et al. 5/609
5,720,059 A * 2/1998 Allevato et al. 5/610

(Continued)

FOREIGN PATENT DOCUMENTS

JP 7-8481 1/1995

Primary Examiner — Peter M Cuomo

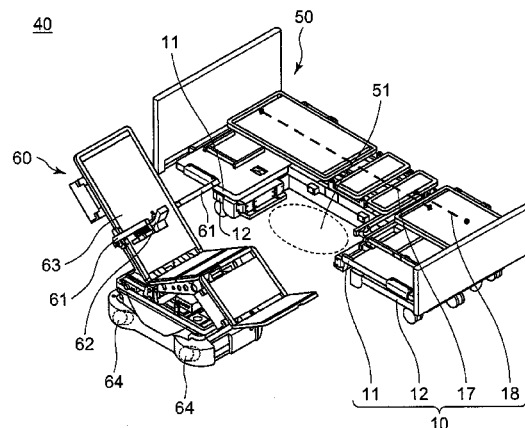
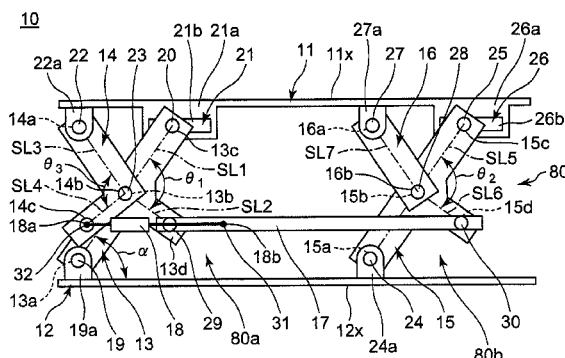
Assistant Examiner — Brittany Wilson

(74) Attorney, Agent, or Firm — Wenderoth, Lind & Ponack, L.L.P.

(57) **ABSTRACT**

A lifting apparatus includes an upper frame, a base frame, a link mechanism for connecting the upper frame and the base frame, and a linear actuator connected to the link mechanism for driving the link mechanism. The link mechanism includes at least a T-shaped first arm slidably supported on the base frame or the upper frame and provided with connections at four points, and an L-shaped second arm provided with connections at three points.

15 Claims, 6 Drawing Sheets



(56)

References Cited

U.S. PATENT DOCUMENTS

6,161,236	A *	12/2000	Carroll	5/618	7,451,505	B2 *	11/2008	Johannsen	5/607
6,493,886	B1 *	12/2002	Vanpage et al.	5/118	7,849,538	B1 *	12/2010	Edgerton	5/611
6,505,362	B1 *	1/2003	Scipio	5/118	8,321,976	B1 *	12/2012	Edgerton	5/611
6,851,144	B2 *	2/2005	Wang	5/610	2004/0055087	A1 *	3/2004	Edgerton	5/611
7,373,677	B2 *	5/2008	Barthelt	5/618	2007/0067912	A1 *	3/2007	Lemire	5/611
					2009/0038074	A1 *	2/2009	Barthelt	5/611
					2010/0017965	A1 *	1/2010	Barthelt	5/611
					2010/0064441	A1 *	3/2010	Barthelt	5/616

* cited by examiner

Fig. 1A

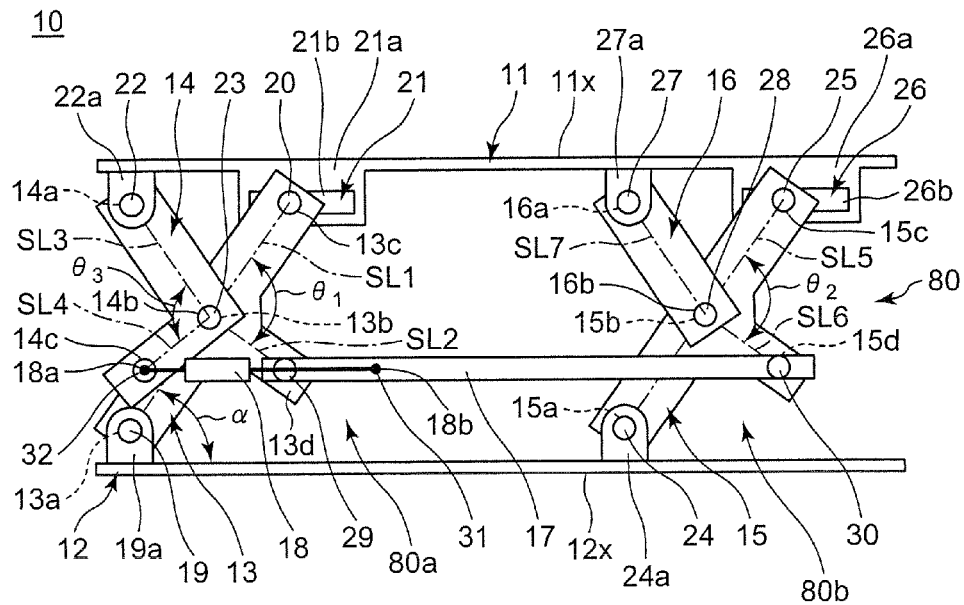


Fig. 1B

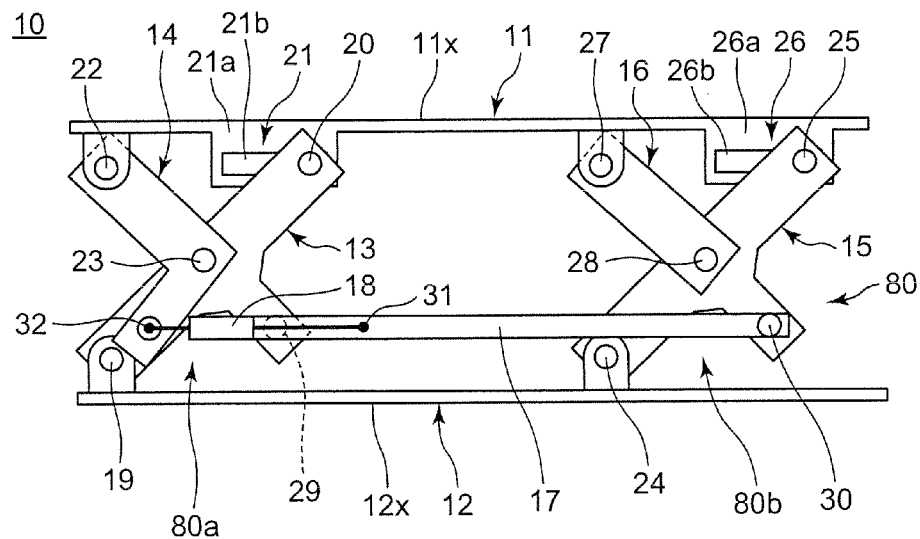


Fig. 2

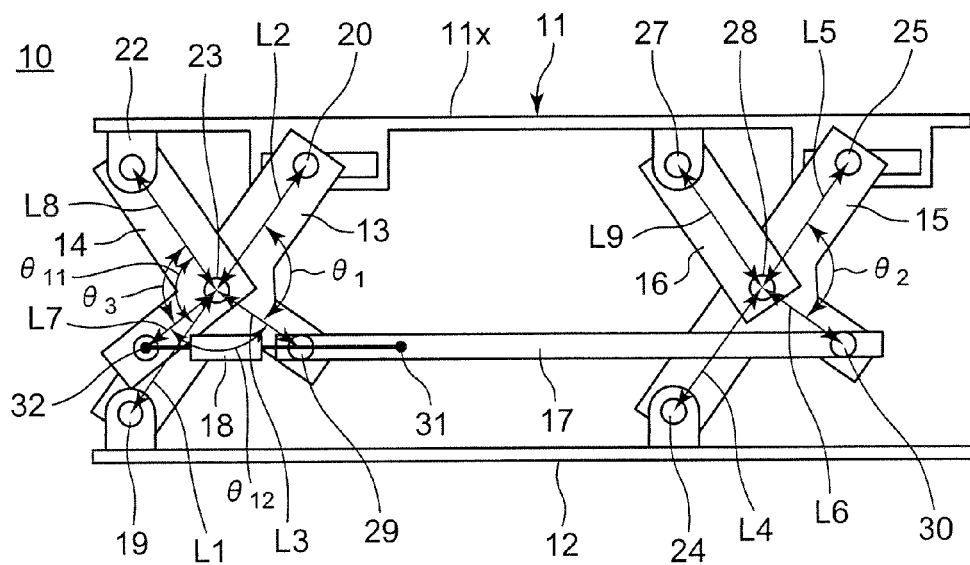


Fig. 3

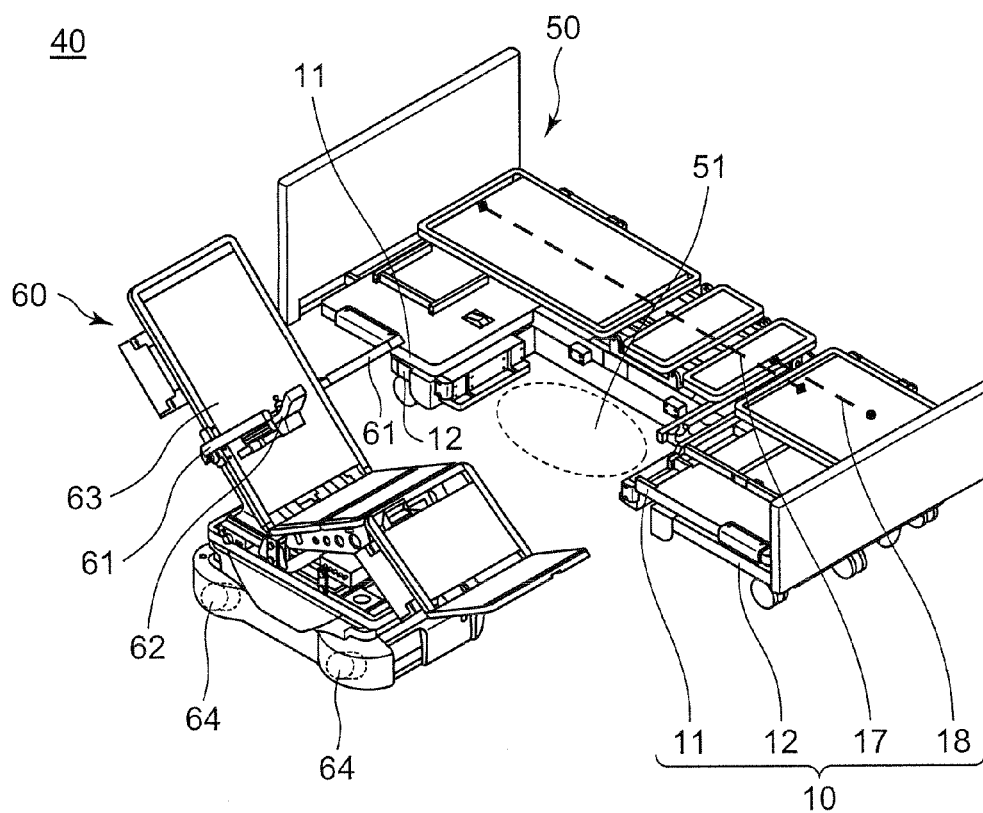


Fig. 4A

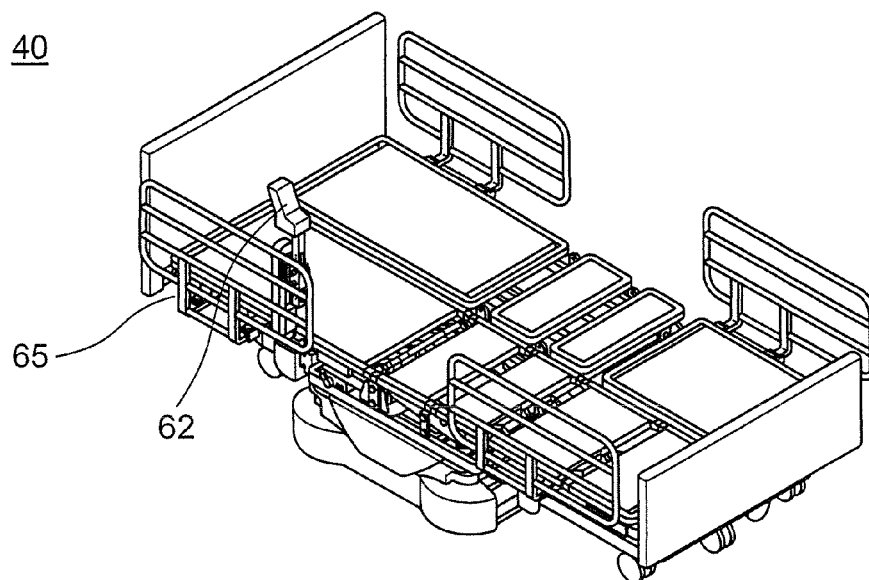


Fig. 4B

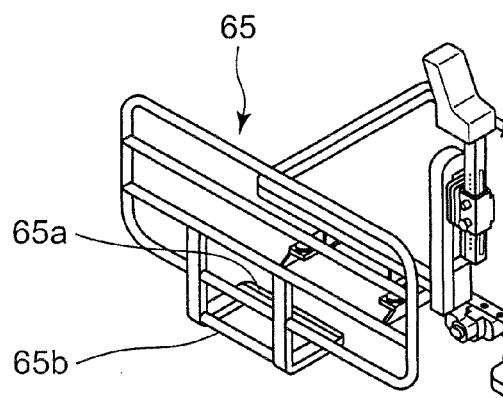


Fig. 4C

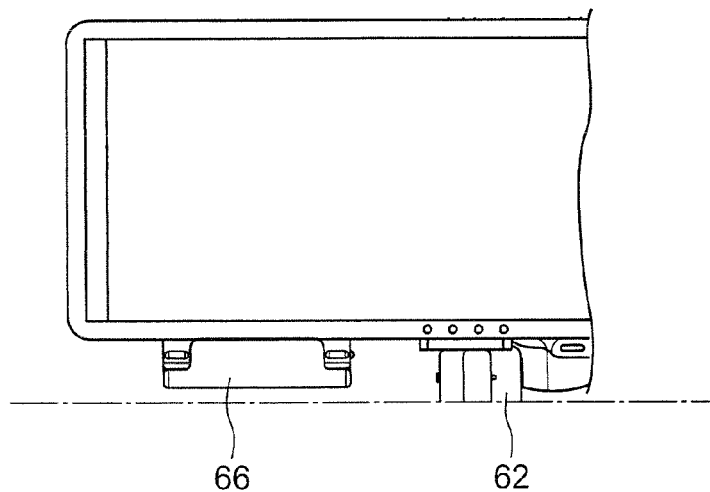


Fig. 5A

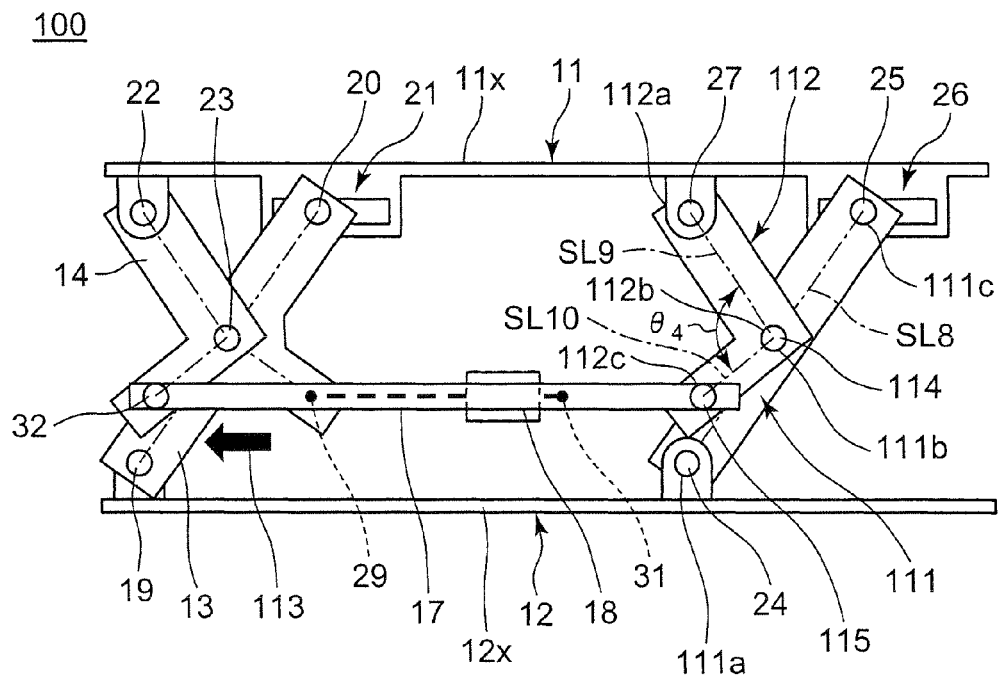


Fig. 5B

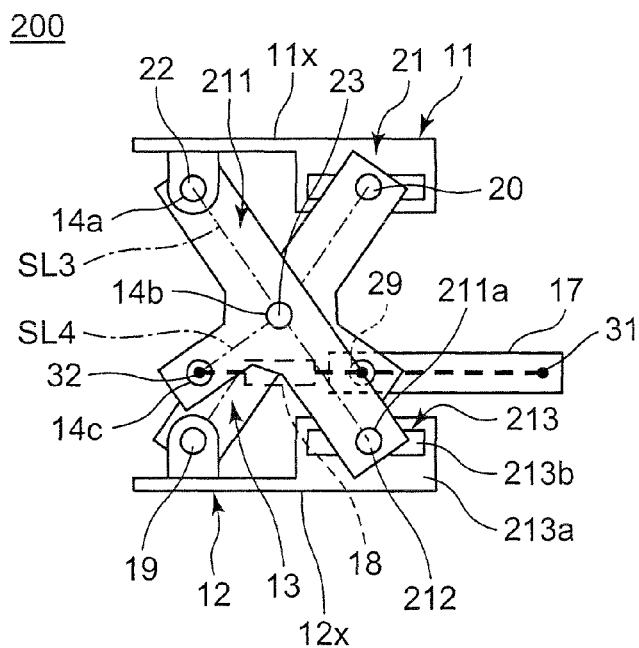
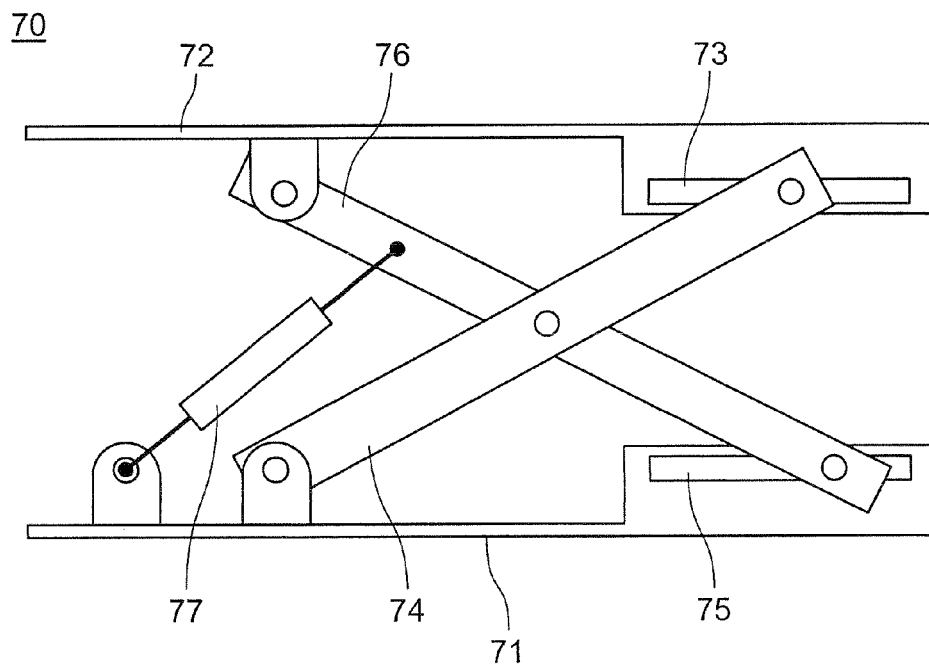


Fig. 6 PRIOR ART

1

LIFTING APPARATUS AND BED PROVIDED WITH THE SAME

BACKGROUND OF THE INVENTION

The present invention relates to a lifting apparatus for converting a linear action by a linear actuator into a linear action in a different direction, and a bed provided with the same.

There is a lifting apparatus for linearly lifting and lowering a heavy item (for example, refer to Patent Literature 1 (JP 7-8481 A)). This lifting apparatus is formed by, for example, a linear actuator including an excellent thrust force and an X-shaped link structure.

FIG. 6 shows a configuration view of the conventional lifting apparatus described in Patent Literature 1.

As shown in FIG. 6, a lifting apparatus 70 is formed by a foundation 71, a top plate 72, linear arms 74, 76, and a linear actuator 77. One end of the linear arm 74 is rotatably connected to the foundation 71, and the other end is connected slidably in a groove 73 of the top plate 72. One end of the linear arm 76 is rotatably connected to the top plate 72, the other end is connected slidably in a groove 75 of the foundation 71, and a center part is rotatably connected to the linear arm 74. One end of the linear actuator 77 is connected to the foundation 71, and the other end is connected to the linear arm 76. This lifting apparatus 70 is a mechanism of lifting and lowering the top plate 72 vertically with respect to the foundation 71.

SUMMARY OF THE INVENTION

An object of the present invention is to provide a lifting apparatus in which a load applied to an actuator is as constant as possible without depending on a distance between an upper frame (top plate) and a base frame (foundation), and a bed provided with the same.

In accomplishing these and other aspects, according to one aspect of the present invention, there is provided a lifting apparatus comprising a link mechanism lifting and lowering an upper frame with respect to a base frame, wherein the link mechanism comprises:

a first arm slidably supported on the base frame or the upper frame, the first arm including first, second, and third connections placed on a first straight line in order, and a fourth connection positioned on a second straight line which is inclined by a predetermined angle with respect to the first straight line from the second connection;

a second arm including fifth and sixth connections, and a seventh connection positioned on a fourth straight line which is inclined by a predetermined angle with respect to a third straight line connecting the fifth and sixth connections, the fourth straight line passing through the sixth connection; and

a fifth arm connected to the fourth connection of the first arm,

the second connection of the first arm and the sixth connection of the second arm are rotatably connected,

a linear actuator has one end connected to the fifth connection and the other end connected to the seventh connection of the second arm, and

by driving the link mechanism with using the linear actuator, the upper frame is relatively lifted and lowered with respect to the base frame.

BRIEF DESCRIPTION OF THE DRAWINGS

These and other aspects and features of the present invention will become clear from the following description taken in

2

conjunction with the embodiments thereof with reference to the accompanying drawings, in which:

FIG. 1A is a configuration view of a lifting apparatus in an embodiment of the present invention;

FIG. 1B is a configuration view of the lifting apparatus in the embodiment of the present invention;

FIG. 2 is a view showing size definition of the lifting apparatus in the embodiment;

FIG. 3 is a perspective view showing a separation state of a separation type bed in which the lifting apparatus of the embodiment is used;

FIG. 4A is a perspective view showing a bed state of the separation type bed in which the lifting apparatus of the embodiment is used;

FIG. 4B is a detailed segmentary view of the bed state of the separation type bed of the embodiment;

FIG. 4C is a segmentary plan view of the bed state of the separation type bed in the embodiment;

FIG. 5A is a view showing a lifting apparatus of a first different configuration in the embodiment;

FIG. 5B is a view showing a lifting apparatus of a second different configuration in the embodiment; and

FIG. 6 is a configuration view of a conventional lifting apparatus.

DESCRIPTION OF THE EMBODIMENTS

Hereinafter, an embodiment of the present invention will be described with reference to the drawings. It should be noted that the same constituent elements will be given the same reference numerals, and description thereof will be omitted in some cases. For easy understanding, the drawings are schematic focusing on the constituent elements.

Embodiment

FIG. 1A is a configuration view of a lifting state of a lifting apparatus 10 according to one embodiment of the present invention. FIG. 1B is a configuration view of a lowering state of the lifting apparatus 10, and FIG. 2 is a view showing size definition of the lifting apparatus 10.

As shown in FIGS. 1A and 1B, the lifting apparatus 10 of the embodiment is a lifting apparatus for lifting and lowering an upper frame 11 with respect to a base frame 12. This lifting apparatus 10 includes at least the base frame 12, the upper frame 11, a linear actuator 18, and a link mechanism 80.

The link mechanism 80 includes a first arm 13, a second arm 14, a third arm 15, a fourth arm 16, and a fifth arm 17 as one example. The first arm 13 is one example of a driving side T-shaped arm, and the second arm 14 is one example of a driving side L-shaped arm. The third arm 15 is one example of a driven side T-shaped arm, the fourth arm 16 is one example of a driven side I-shaped arm, and the fifth arm 17 is one example of a rod shape conjunction arm.

The upper frame 11 includes a first protruding portion 22a fixed to a front end thereof, a second protruding portion 21a fixed in the vicinity of the first protruding portion 22a, a third protruding portion 27a fixed to a portion behind the second protruding portion 21a, and a fourth protruding portion 26a fixed in the vicinity of a portion behind the third protruding portion 27a and in a rear end of the upper frame. The second protruding portion 21a has a groove 21b extending in the longitudinal direction parallel to the upper frame 11. By disengageably engaging a first pivot point 20 to be described later into the groove 21b so that the first pivot point 20 can move slidably in the groove 21b, a slide guide 21 is formed. The first pivot point 20 is, for example, a driving side upper

3

slide pivot point (support axis). The fourth protruding portion 26a has a groove 26b extending in the longitudinal direction parallel to the upper frame 11. By disengageably engaging a seventh pivot point 25 to be described later into the groove 26b so that the seventh pivot point 25 can move slidably in the groove 26b, a slide guide 26 is formed. The seventh pivot point 25 is, for example, a driven side upper slide pivot point (support axis).

The base frame 12 is arranged so as to face the upper frame 11. The base frame 12 includes a fifth protruding portion 19a fixed so as to face the first protruding portion 22a of the upper frame 11, and a sixth protruding portion 24a fixed so as to face the third protruding portion 27a of the upper frame 11. Therefore, a mounting surface 12x of the base frame 12 and a loading surface 11x of the upper frame 11 are arranged so as to be vertical to a straight line connecting a second pivot point 19 and a third pivot point 22 to be described later. The mounting surface 12x of the base frame 12 and the loading surface 11x of the upper frame 11 are arranged so as to be vertical to a straight line connecting an eighth pivot point 24 and a ninth pivot point 27 to be described later. The second pivot point 19 is, for example, a driving side lower fixed pivot point (support axis). The third pivot point 22 is, for example, a driving side upper fixed pivot point (support axis). The eighth pivot point 24 is, for example, a driven side lower fixed pivot point (support axis). The ninth pivot point 27 is, for example, a driven side upper fixed pivot point (support axis). The loading surface 11x is, for example, a lifting item contact surface for loading a lifting item.

As shown in FIGS. 1A and 1B, when seen from a side surface, the upper frame 11 is arranged in parallel with the base frame 12. In detail, when seen from the side surface, the loading surface 11x of the upper frame 11 is arranged in parallel with the mounting surface 12x of the base frame 12.

The link mechanism 80 is arranged between the upper frame 11 and the base frame 12. The link mechanism 80 is formed by various arms (the first arm 13, the second arm 14, the third arm 15, the fourth arm 16, and the fifth arm 17). This link mechanism 80 functions as one example of a link mechanism, and lifts and lowers the upper frame 11 with respect to the base frame 12.

The link mechanism 80 is formed by a first link mechanism 80a and a second link mechanism 80b. The first link mechanism 80a is, for example, a driving side link mechanism, and the second link mechanism 80b is, for example, a driven side link mechanism.

The first link mechanism 80a is formed by at least the first arm 13 and the second arm 14. The second link mechanism 80b is formed by at least the third arm 15 and the fourth arm 16. The first arm 13 and the third arm 15 each are one example of a T-shaped arm. The second arm 14 is one example of an L-shaped arm. The fourth arm 16 is one example of an I-shaped arm.

The T-shaped arm is provided with connections at four points. The connections at four points of the T-shaped arm are connections at three points placed on a first straight line, and an offset connection at one point positioned on a second straight line which is inclined by a predetermined angle with respect to the first straight line from the center connection.

Connections at four points of the first arm 13 are a first connection 13a, a second connection 13b, a third connection 13c, and a fourth connection 13d. Specifically, these connections at four points are connections at three points placed on the first straight line SL1 between an upper end part and a lower end part of the first arm 13 (the first connection 13a, the second connection 13b, and the third connection 13c), and an offset connection (the fourth connection 13d) at one point

4

positioned on a second straight line SL2 which is inclined by a predetermined angle $\theta 1$ with respect to the first straight line SL1 from the center second connection 13b. The first connection 13a is the second pivot point 19, the second connection 13b is a fourth pivot point 23, the third connection 13c is the first pivot point 20, and the fourth connection 13d is a fifth pivot point 29.

Connections at four points of the third arm 15 are an eighth connection 15a, a ninth connection 15b, a tenth connection 15c, and an eleventh connection 15d. Specifically, these connections at four points are connections at three points placed on a first straight line SL5 between an upper end part and a lower end part of the third arm 15 (the eighth connection 15a, the ninth connection 15b, and the tenth connection 15c), and an offset connection (the eleventh connection 15d) at one point positioned on a second straight line SL6 which is inclined by a predetermined angle $\theta 2$ with respect to the first straight line SL5 from the center ninth connection 15b. The eighth connection 15a is the eighth pivot point 24, the ninth connection 15b is a tenth pivot point 28, the tenth connection 15c is the seventh pivot point 25, and the eleventh connection 15d is an eleventh pivot point 30.

The L-shaped arm is provided with connections at three points. The connections at three points of the L-shaped arm are connections at two points, and an offset connection at one point positioned on a fourth straight line which is inclined by a predetermined angle with respect to a third straight line connecting the connections at two points.

Connections at three points of the second arm 14 are a fifth connection 14a, a sixth connection 14b, and a seventh connection 14c. Specifically, these connections at three points are connections (the fifth connection 14a and the sixth connection 14b) at two points placed on a third straight line SL3 between an upper end part and an intermediate part of the second arm 14, and an offset connection (the seventh connection 14c) at one point positioned on a fourth straight line SL4 which is inclined by a predetermined angle $\theta 3$ with respect to the third straight line SL3. The fifth connection 14a is the third pivot point 22, the sixth connection 14b is the fourth pivot point 23, and the seventh connection 14c is a sixth pivot point 32.

It should be noted that the L-shaped arm can function as a T-shaped arm by extending a part thereof. Therefore, the L-shaped arm in this embodiment includes the T-shaped arm. For example, although described in detail later, an eighth arm 211 of FIG. 5B is a modification of the second arm 14.

The I-shaped arm is provided with connections at two points in both ends of the longitudinal direction.

The fourth arm 16 is provided with connections at two points in both ends in the longitudinal direction, that is, a twelfth connection 16a in an upper end part, and a thirteenth connection 16b in a lower end part. The twelfth connection 16a is the ninth pivot point 27, and the thirteenth connection 16b is the tenth pivot point 28.

The first connection 13a serving as the lower end part of the first arm 13 is rotatably connected to the fifth protruding portion 19a of the base frame 12 at the second pivot point 19. The third connection 13c serving as the upper end part of the first arm 13 is connected to the upper frame 11 slidably by the slide guide 21. Specifically, by disengageably engaging the first pivot point 20 into the groove 21b of the second protruding portion 21a of the upper frame 11, the third connection 13c of the first arm 13 is slidably connected to the upper frame 11.

The fifth connection 14a serving as the upper end part of the second arm 14 is rotatably connected to the first protruding portion 22a of the upper frame 11 at the third pivot point

5

22. The sixth connection **14b** of the second arm **14** is rotatably connected to the second connection **13b** of the first arm **13** at the fourth pivot point **23**. The fourth pivot point **23** is, for example, a driving side arm connection pivot point (support axis).

The eighth connection **15a** serving as the lower end part of the third arm **15** is rotatably connected to the sixth protruding portion **24a** of the base frame **12** at the eighth pivot point **24**. The tenth connection **15c** serving as the upper end part of the third arm **15** is connected to the upper frame **11** slidably by the slide guide **26**. Specifically, by disengageably engaging the seventh pivot point **25** into the groove **26b** of the fourth protruding portion **26a** of the upper frame **11**, the tenth connection **15c** serving as the upper end part of the third arm **15** is slidably connected to the upper frame **11**.

The twelfth connection **16a** serving as the upper end part of the fourth arm **16** is rotatably connected to the third protruding portion **27a** of the upper frame **11** at the ninth pivot point **27**. The thirteenth connection **16b** serving as the lower end part of the fourth arm **16** is rotatably connected to the ninth connection **15b** of the third arm **15** at the tenth pivot point **28**. The tenth pivot point **28** is, for example, a driven side arm connection pivot point (support axis).

A driving side end part of the fifth arm **17** is rotatably connected to the fourth connection **13d** of the first arm **13** at the fifth pivot point **29**. The fifth pivot point **29** is, for example, a driving side lower coupling arm pivot point (support axis). A driven side end part of the fifth arm **17** is rotatably connected to the eleventh connection **15d** of the third arm **15** at the eleventh pivot point **30**. The eleventh pivot point **30** is, for example, a driven side lower coupling arm pivot point (support axis). An end part **18b** of the linear actuator **18** is rotatably connected to an actuator fixing portion **31** of the fifth arm **17** in the vicinity of the driving side end part. The actuator fixing portion **31** is positioned between the fifth pivot point **29** and the eleventh pivot point **30**.

One end part **18b** of the linear actuator **18** is rotatably connected to the actuator fixing portion **31** on the fifth arm **17**, and the other end part **18a** is rotatably connected to the seventh connection **14c** of the second arm **14** at the sixth pivot point **32**. The sixth pivot point **32** is, for example, a driving side lower coupling arm pivot point (support axis). The linear actuator **18** is connected to any arms of the link mechanism **80** (the first arm **13**, the second arm **14**, the third arm **15**, the fourth arm **16**, and the fifth arm **17**), so as to drive to lift and lower the upper frame **11** with respect to the base frame **12**.

As shown on the left side of FIG. 2, a distance between center of the second pivot point **19** and center of the fourth pivot point **23** is defined as **L1**. A distance between center of the first pivot point **20** and the center of the fourth pivot point **23** is defined as **L2**. A distance between center of the fifth pivot point **29** and the center of the fourth pivot point **23** is defined as **L3**. A distance between center of the sixth pivot point **32** and the center of the fourth pivot point **23** is defined as **L7**. A distance between center of the third pivot point **22** and the center of the fourth pivot point **23** is defined as **L8**. A predetermined angle made by a straight line (part of the first straight line **SL1**) connecting the center of the first pivot point **20** and the center of the fourth pivot point **23** and a straight line (the second straight line **SL2**) connecting the center of the fifth pivot point **29** and the center of the fourth pivot point **23** is defined as $\theta 1$. A predetermined angle made by a straight line (the third straight line **SL3**) connecting the center of the sixth pivot point **32** and the center of the fourth pivot point **23** and a straight line (the fourth straight line **SL4**) connecting the center of the third pivot point **22** and the center of the fourth pivot point **23** is defined as $\theta 3$. An angle made by a straight

6

line connecting the third pivot point **22** and the fourth pivot point **23** and a straight line connecting the second pivot point **19** and the fourth pivot point **23** is defined as $\theta 11$. An angle made by a straight line connecting the fifth pivot point **29** and the fourth pivot point **23** and a straight line connecting the sixth pivot point **32** and the fourth pivot point **23** is defined as $\theta 12$. It should be noted that in the embodiment, the second pivot point **19**, the first pivot point **20**, and the fourth pivot point **23** are arranged on the same straight line (the first straight line **SL1**).

Further, as shown on the right side of FIG. 2, a distance between center of the eighth pivot point **24** and center of the tenth pivot point **28** is defined as **L4**. A distance between center of the seventh pivot point **25** and the center of the tenth pivot point **28** is defined as **L5**. A distance between center of the eleventh pivot point **30** and the center of the tenth pivot point **28** is defined as **L6**. A distance between center of the ninth pivot point **27** and the center of the tenth pivot point **28** is defined as **L9**. A predetermined angle made by a straight line (part of the first straight line **SL5**) connecting the center of the seventh pivot point **25** and the center of the tenth pivot point **28** and a straight line (the second straight line **SL6**) connecting the center of the eleventh pivot point **30** and the center of the tenth pivot point **28** is defined as $\theta 2$. It should be noted that in the embodiment, the center of the eighth pivot point **24**, the center of the seventh pivot point **25**, and the center of the tenth pivot point **28** are arranged on the same straight line (the first straight line **SL5**).

When the inventors variously examined regarding the lifting apparatus **10** formed in such a way, it was found that by setting the configuration of the link mechanism **80** under a predetermined condition, a load applied to the linear actuator **18** by a heavy item loaded on the loading surface **11x** of the upper frame **11** becomes substantially constant without depending on a position (height) of the upper frame **11**. Specifically, it was found that with a triangular-shape formed by the fourth pivot point **23**, the second pivot point **19**, and the third pivot point **22**, and a triangular-shape formed by the fourth pivot point **23**, the fifth pivot point **29**, and the sixth pivot point **32** as similar shapes in FIG. 2, the load applied to the linear actuator **18** becomes substantially constant without depending on the position (height) of the upper frame **11**. Further specifically, it was found that with $L1=L8$ and $L3=L7$ and the angle $\theta 11=\theta 12$ in FIG. 2, the load applied to the linear actuator **18** becomes substantially constant without depending on the position (height) of the upper frame **11**. It should be noted that at this time, in the link mechanism **80**, $L1=L2=L8=L4=L5=L9$, $L3=L7=L6$, and the angle $\theta 1=\theta 2=\theta 3=90^\circ$. In the link mechanism **80** of the embodiment, the triangular-shape formed by the fourth pivot point **23**, the second pivot point **19**, and the third pivot point **22**, and a triangular-shape formed by the tenth pivot point **28**, the ninth pivot point **27**, and the eighth pivot point **24** have the same shape.

Further, it was found that at this time, as the angle $\theta 1 (= \theta 3)$ is more different from 90° , the load applied to the linear actuator **18** in accordance with the position of the upper frame **11** is changed more.

It should be noted that the load applied to the linear actuator **18** is obtained by multiplying the load by the heavy item loaded on the loading surface **11x** of the upper frame **11** by a constant calculated by $L1/L3$.

In the embodiment, the triangular-shape formed by the fourth pivot point **23**, the sixth pivot point **32**, and the fifth pivot point **29**, and the triangular-shape formed by the third pivot point **22**, the fourth pivot point **23**, and the second pivot point **19** are isosceles triangular-shapes having an apex angle

7

of 2α ($=\theta_{11}=\theta_{12}$) which are similar shapes. The angle α is an angle made by the loading surface **11x** of the upper frame **11** or the mounting surface **12x** of the base frame **12** and **L1** or **L2** (the first straight line **SL1**).

The linear actuator **18** is arranged so as to connect the sixth pivot point **32** and the fifth pivot point **29**. A change amount of a distance between the sixth pivot point **32** and the fifth pivot point **29** is a drive amount of the linear actuator **18**. A distance between the third pivot point **22** and the second pivot point **19** is lifting height, and a change amount of the distance between the third pivot point **22** and the second pivot point **19** is a lifting amount. Therefore, a ratio between the drive amount of the linear actuator **18** and the lifting amount is $L1/L3$. As a result, the load applied to this lifting apparatus **10** becomes always constant without depending on the angle α .

It should be noted that in the embodiment, positional relationships between the pivot points of the third arm **15** are the same as the first arm **13** with $L1=L4$, $L2=L5$, $L3=L6$, and $\theta_1=\theta_2$. Regarding the second arm **14** and the fourth arm **16**, $L8=L9$.

FIG. 3 is a perspective view of a separation type bed **40** in which the lifting apparatus **10** of the embodiment is used. FIG. 4A is a perspective view of the time when the separation type bed **40** is deformed into a bed state. FIG. 4B is an enlarged segmentary view in the perspective view of the separation type bed **40**. FIG. 4C is an enlarged segmentary view in a plan view of the separation type bed **40**.

As shown in FIG. 3, the separation type bed **40** is formed by a bed unit **50** and a wheelchair unit **60**.

The bed unit **50** includes the above lifting apparatus **10** inside thereof, and is formed in such a manner that a support member in an upper part of the bed unit **50** is lifted and lowered by the lifting apparatus **10** with respect to a base of the bed unit **50**. The support member of the bed unit **50** is a member for supporting a mattress or the like on which a user lies. The wheelchair unit **60** is stored in a storage space **51** provided in a part on one side in the width direction in the bed unit **50** so as to form the bed state. In this bed state, the wheelchair unit **60** is lifted and lowered together with the upper frame **11** by the lifting apparatus **10**. In the bed unit **50**, the fifth arm **17** and the linear actuator **18** are arranged only on one side (on the far side in FIG. 3) in the width direction of the bed unit **50** but not arranged on the other side (on the near side in FIG. 3) in the width direction of the bed unit **50** where the storage space **51** is provided. With such a configuration, the large storage space **51** is ensured in the bed unit **50**.

The wheelchair unit **60** includes two armrests **61**, an operation unit **62** provided in a front end of one of the armrests **61** (on the side opposite to the bed unit **50** at the time of combination), a seat unit **63** to be deformed from a chair form into a flat form by an input of the operation unit **62**, and four wheels **64** for moving the wheelchair unit **60**.

The separation type bed **40** of the embodiment can be deformed from a separated state in which the bed unit **50** and the wheelchair unit **60** are separated into the bed state in which the bed unit **50** and the wheelchair unit **60** are combined and the entire surfaces of the bed unit **50** and the wheelchair unit **60** are brought into a flat form, by moving the wheelchair unit **60** to the storage space **51** by the input of the operation unit **62** by an operator, and then bringing the seat unit **63** into a flat form. As shown in FIG. 4A, the separation type bed **40** deformed into the bed state integrally lifts and lowers the bed unit **50** and the wheelchair unit **60** at the same time by the lifting apparatus **10** provided inside thereof.

Side rails **65** are installed on side surfaces of the separation type bed **40**, and the user can be prevented from falling from the side surfaces of the separation type bed **40** by the side rails

8

65. As shown in FIGS. 4B and 4C, the side rails **65** are inserted into side rail holders **66** provided in the bed unit **50** and the wheelchair unit **60** from the side.

FIG. 5A is a configuration view of a lifting apparatus **100** serving as a different configuration of the lifting apparatus **10** and a first different configuration of the embodiment. FIG. 5B is a configuration view of a lifting apparatus **200** serving as a different configuration of the lifting apparatus **10** and a second different configuration of the embodiment.

As shown in FIG. 5A, the lifting apparatus **100** serving as the first different configuration of the embodiment includes an upper frame **11**, a base frame **12**, a first arm **13**, a second arm **14**, a sixth arm **111**, a seventh arm **112**, a fifth arm **17**, and a linear actuator **18**. That is, the lifting apparatus **100** serving as the first different configuration has a different driven side arm configuration from the above lifting apparatus **10**. The second arm **14** and the seventh arm **112** are respectively one example of the L-shaped arm, and the sixth arm **111** is one example of the I-shaped arm. The first arm **13** and the second arm **14** are a first link mechanism. The sixth arm **111** and the seventh arm **112** are a second link mechanism. The sixth arm **111** has a fourteenth connection **111a**, a fifteenth connection **111b**, and a sixteenth connection **111c** in order on one straight line **SL8** obliquely upward from the lower side. The seventh arm **112** has a seventeenth connection **112a** and an eighteenth connection **112b** arranged in order obliquely downward from the upper side, and a nineteenth connection **112c** positioned on a straight line **SL10** which is inclined by a predetermined angle θ_4 with respect to a straight line **SL9** connecting the seventeenth connection **112a** and the eighteenth connection **112b**, the straight line passing **SL10** through the eighteenth connection **112b**. The angle θ_4 is the same angle as the angle θ_3 .

The fifteenth connection **111b** of the sixth arm **111** and the eighteenth connection **112b** of the seventh arm **112** are rotatably connected at a fourteenth pivot point **114**. In the lifting apparatus **100**, a sixth pivot point **32** of the second arm **14** and a thirteenth pivot point **115** of the seventh arm **112** are connected by the fifth arm **17**. An actuator fixing portion **31** on the fifth arm **17** and a fifth pivot point **29** of the first arm **13** are connected by the linear actuator **18**.

In the lifting apparatus **100** of the first different configuration, lengths of the arms are the same i.e. the straight line **SL8**=the straight line **SL5**, the straight line **SL9**=the straight line **SL7**, and the straight line **SL10**=the straight line **SL4**. Therefore, a triangular-shape formed by a fourth pivot point **23**, a second pivot point **19**, and a third pivot point **22**, and a triangular-shape formed by the fourteenth pivot point **114**, a ninth pivot point **27**, and an eighth pivot point **24** have the same shape. A triangular-shape formed by the fourth pivot point **23**, the sixth pivot point **32**, and the third pivot point **22**, and a triangular-shape formed by the fourteenth pivot point **114**, the ninth pivot point **27**, and the thirteenth pivot point **115** have the same shape.

Regarding the fifth pivot point **29** or the sixth pivot point **32**, the direction of the pivot point not connected to the linear actuator **18** with respect to the pivot point connected to the linear actuator **18** is defined as a determination direction **113**. By arranging the actuator fixing portion **31** of the lifting apparatus **100** in the opposite direction to the determination direction **113** with respect to the pivot point connected to the linear actuator **18**, the direction of a load applied to the linear actuator **18** in the lifting apparatus **100** is reversed from the load applied to the linear actuator **18** in the above lifting apparatus **10**, so that a tensile load is added. As a result, the lifting apparatus **100** of the first different configuration is

particularly effective in a case where the linear actuator **18** has a favorable characteristic with respect to tension.

As shown in FIG. 5B, the lifting apparatus **200** serving as the second different configuration of the embodiment is formed by an upper frame **11**, a base frame **12**, a first arm **13**, an eighth arm **211**, a fifth arm **17**, and a linear actuator **18**. That is, the lifting apparatus **200** serving as the second different configuration has only a driving side arm configuration without a driven side arm configuration, and has a different driving side arm configuration from the above lifting apparatus **10**. Each of the first arm **13** and the eighth arm **211** is one example of the T-shaped arm. The eighth arm **211** is the modification of the second arm **14**.

In the lifting apparatus **200**, the eighth arm **211** is connected to other members at a third pivot point **22**, a twelfth pivot point **212**, a fourth pivot point **23**, and a sixth pivot point **32**. Specifically, the eighth arm **211** is rotatably coupled to the upper frame **11** at the third pivot point **22**, slidably connected to a slide guide **213** provided in the base frame **12** at the twelfth pivot point **212**, and rotatably coupled to the first arm **13** at the fourth pivot point **23**. The slide guide **213** has a similar structure to the slide guide **21**. That is, the base frame **12** has a fixed protruding portion **213a**, the protruding portion **213a** has a groove **213b** extending in the longitudinal direction, and the twelfth pivot point **212** is disengageably engaged into the groove **213b** so as to be slidably movable, thereby forming the slide guide **213**. The twelfth pivot point **212** is, for example, a driving side lower slide pivot point (support axis). The fifth arm **17** is rotatably connected to a fifth pivot point **29** of the first arm **13**.

In the lifting apparatus **200** of the second different configuration, a triangular-shape formed by the fourth pivot point **23**, a second pivot point **19**, and the third pivot point **22**, and a triangular-shape formed by the fourth pivot point **23**, a fifth pivot point **29**, and the sixth pivot point **32** are similar shapes. In the second different configuration, when the sum of an angle ($\theta 1$) made by a segment (first straight line) connecting the second pivot point **19** and a first pivot point **20** and a segment (second straight line) connecting the fourth pivot point **23** and the fifth pivot point **29**, and an angle ($\theta 3$) made by a segment (third straight line) connecting the third pivot point **22** and the twelfth pivot point **212** and a segment (fourth straight line) connecting the fourth pivot point **23** and the sixth pivot point **32** is 180° , a load applied to the linear actuator **18** by a heavy item loaded on a loading surface **11x** of the upper frame **11** can be substantially constant without depending on a distance between the upper frame **11** and the base frame **12**. That is, in the second different configuration, with $\theta 1 + \theta 3 = 180^\circ$, the load applied to the linear actuator **18** can be constant.

In the lifting apparatus **200**, by connecting the linear actuator **18** to the sixth pivot point **32** of the eighth arm **211** and an actuator fixing portion **31** of the fifth arm **17**, the load applied to the linear actuator **18** by the heavy item loaded on the loading surface **11x** of the upper frame **11** can be substantially constant without depending on the distance between the upper frame **11** and the base frame **12**.

According to the above embodiment, the lifting apparatus in which the load applied to the linear actuator **18** becomes constant without depending on the distance between the upper frame **11** and the base frame **12**, and the bed provided with the same can be provided.

By appropriately combining arbitrary embodiments or modifications among the above various embodiments and modifications, effects provided in the embodiments or the modifications can be obtained.

It should be noted that although the example of the bed provided with the lifting apparatus is described in the embodiment, the lifting apparatus of the present invention can be utilized for various devices provided with a lifting device in addition to the bed.

The lifting apparatus according to the present invention and the bed provided with the same are particularly useful for a nursing care bed for lifting and lowering a care-receiver within a wide range, and useful in an ordinary house, a hospital facility, or a nursing care facility where the care-receiver in need of care resides.

Although the present invention has been fully described in connection with the embodiments thereof with reference to the accompanying drawings, it is to be noted that various changes and modifications are apparent to those skilled in the art. Such changes and modifications are to be understood as included within the scope of the present invention as defined by the appended claims unless they depart therefrom.

What is claimed is:

1. A lifting apparatus comprising a link mechanism lifting and lowering an upper frame with respect to a base frame, wherein

the link mechanism comprises:

a first arm slidably supported on the base frame or the upper frame, the first arm including first, second, and third connections placed on a first straight line in order, and a fourth connection positioned on a second straight line which is inclined by a predetermined angle with respect to the first straight line from the second connection;

a second arm including fifth and sixth connections, and a seventh connection positioned on a fourth straight line which is inclined by a predetermined angle with respect to a third straight line connecting the fifth and sixth connections, the fourth straight line passing through the sixth connection; and

a fifth arm connected to the fourth connection of the first arm,

the second connection of the first arm and the sixth connection of the second arm are rotatably connected,

a linear actuator has one end connected to the fifth connection and the other end connected to the seventh connection of the second arm,

by driving the link mechanism with using the linear actuator, the upper frame is relatively lifted and lowered with respect to the base frame,

the link mechanism further comprises:

a first link mechanism on a driving side connected to the linear actuator, the first link mechanism including the first arm and the second arm; and

a second link mechanism on a driven side,

the fifth arm connects the first link mechanism and the second link mechanism, and

the second link mechanism comprises:

a T-shaped third arm slidably supported on the base frame or the upper frame, the third arm including eighth, ninth, and tenth connections placed on a fifth straight line in order, and an eleventh connection positioned on a sixth straight line which is inclined by a predetermined angle with respect to the fifth straight line from the ninth connection; and

an I-shaped fourth arm including twelfth and thirteenth connections, and

the ninth connection of the third arm and the thirteenth connection of the fourth arm are rotatably connected.

11

2. The lifting apparatus according to claim 1, wherein a triangular-shape formed by the second connection, the fifth connection, and the first connection, and a triangular-shape formed by the second connection, the fourth connection, and the seventh connection are similar shapes.
3. The lifting apparatus according to claim 2, wherein the first arm is a T-shaped arm, and the second arm is an L-shaped arm.
4. The lifting apparatus according to claim 2, wherein the second connection is positioned in center of a segment connecting the first connection and the third connection.
5. The lifting apparatus according to claim 2, wherein the second link mechanism comprises:
 - a sixth arm including fourteenth, fifteenth, and sixteenth connections; and
 - a seventh arm including seventeenth and eighteenth connections, and a nineteenth connection positioned on a seventh straight line which is inclined by a predetermined angle with respect to an eighth straight line connecting the seventeenth and eighteenth connections, the seventh straight line passing through the eighteenth connection, and
 - the fifteenth connection of the sixth arm and the eighteenth connection of the seventh arm are rotatably connected.
6. The lifting apparatus according to claim 2, wherein the link mechanism comprises the first arm, and a T-shaped sixth arm formed on the third straight line connecting the fifth and sixth connections in the second arm and extended so as to have another connection on an opposite side of the fifth connection with respect to the sixth connection.
7. The lifting apparatus according to claim 2, wherein the predetermined angle is 90° .
8. The lifting apparatus according to claim 2, wherein a sum of an angle made by the first straight line and the second straight line, and an angle made by the third straight line and the fourth straight line is 180° .
9. A bed, comprising:
 - the lifting apparatus according to claim 2; and
 - a bed unit to be lifted and lowered by the lifting apparatus.
10. The lifting apparatus according to claim 1, wherein the first arm is a T-shaped arm, and the second arm is an L-shaped arm.

12

11. The lifting apparatus according to claim 1, wherein the second connection is positioned in center of a segment connecting the first connection and the third connection.
12. The lifting apparatus according to claim 1, wherein the predetermined angle is 90° .
13. The lifting apparatus according to claim 1, wherein a sum of an angle made by the first straight line and the second straight line, and an angle made by the third straight line and the fourth straight line is 180° .
14. A bed, comprising:
 - the lifting apparatus according to claim 1; and
 - a bed unit to be lifted and lowered by the lifting apparatus.
15. A lifting apparatus according to claim 1, comprising a link mechanism lifting and lowering an upper frame with respect to a base frame, wherein the link mechanism comprises:
 - a first arm slidably supported on the base frame or the upper frame, the first arm including first, second, and third connections placed on a first straight line in order, and a fourth connection positioned on a second straight line which is inclined by a predetermined angle with respect to the first straight line from the second connection;
 - a second arm including fifth and sixth connections, and a seventh connection positioned on a fourth straight line which is inclined by a predetermined angle with respect to a third straight line connecting the fifth and sixth connections, the fourth straight line passing through the sixth connection; and
 - a third arm connected to the fourth connection of the first arm,
- the second connection of the first arm and the sixth connection of the second arm are rotatably connected,
- a linear actuator has one end connected to the fifth connection and the other end connected to the seventh connection of the second arm,
- by driving the link mechanism with using the linear actuator, the upper frame is relatively lifted and lowered with respect to the base frame, and
- the link mechanism comprises the first arm, and a T-shaped fourth arm formed on the third straight line connecting the fifth and sixth connections in the second arm and extended so as to have another connection on an opposite side of the fifth connection with respect to the sixth connection.

* * * * *